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**High Strength  
Structural  
Bolting  
Assemblies  
for Preloading  
BS EN 14399-3  
PC 10.9**

## Use of High Strength Structural Bolting Assemblies for Preloading BS EN 14399-3 PC 10.9 Hex with Direct Tension Indicators (DTI's)

### Assembly Configuration – DTI fitted under the nut

The standard assembly configuration for BS EN 14399-3 property class 10.9 hexagon head bolting assemblies is when the DTI is fitted under the nut with a nut face washer and the assembly is tightened by nut rotation, as follows:-

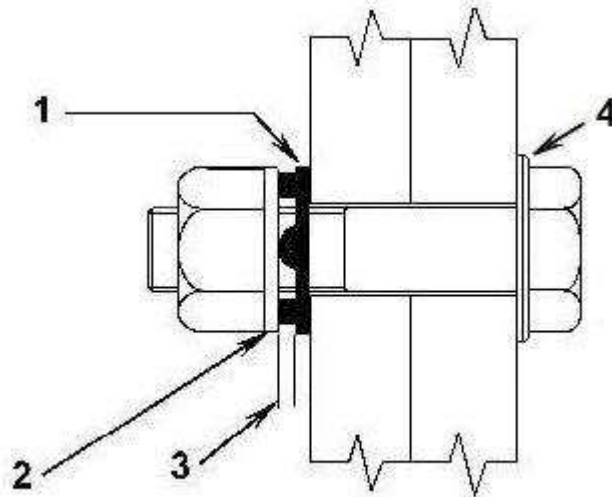
- a) BS EN 14399-3 property class 10.9 hexagon head bolt (marked 10.9 HR)
- b) BS EN 14399-6 hardened chamfered washer (marked H) (component identified as 4 below)
- c) BS EN 14399-9 Direct tension indicator (marked H10) (component identified as 1 below)
- d) BS EN 14399-9 nut face washer (marked HN) (component identified as 2 below)
- e) BS EN 14399-3 property class 10 nut (marked 10 HR)

The components shall be assembled as shown below (Figure 1) with the hardened chamfered washer fitted under the bolt head with the chamfered face of the washer in contact with the under side of the bolt head.

Note: The washers have a chamfer on the inside diameter with a corresponding chamfer (on the same side of the washer) on the outside diameter.

The bolt and washer are placed in the steelwork and at the other side of the connection the direct tension indicator is fitted with the flat side placed against the steelwork and the indicator protrusions facing outwards. The nut face washer is fitted over the bolt threads so that it sits on top of the indicator protrusions and the nut is assembled with the side containing the marking facing outwards and the smooth unmarked side of the nut against the nut face washer.

In this configuration the indicator protrusions shall always bear against the nut face washer – **NO OTHER WAY WILL DO**



**KEY**

- 1 Direct Tension Indicator
- 2 Nut face washer
- 3 Gap
- 4 Washer according to EN 14399-6

**Assembly configuration for EN 14399-3 Property Class 10.9  
with DTI fitted under the nut - Tightened by nut rotation  
Figure 1**

**Initial Tightening**

The bolt head shall be prevented from rotation and each assembly shall be brought to 'snug tight' condition by nut rotation. The tightening process shall be carried out from bolt to bolt within the group, starting from the most rigid part of the connection and moving progressively towards the least rigid part.

Note: The most rigid part of a cover plate connection of an 'I' section is commonly in the middle of the connection bolt group.

The 'snug tight' condition of a fastener assembly shall be when initial deformation of the DTI protrusions begins. This first step shall be completed for all bolts in one connection prior to commencement of the second step.

**Final Tightening**

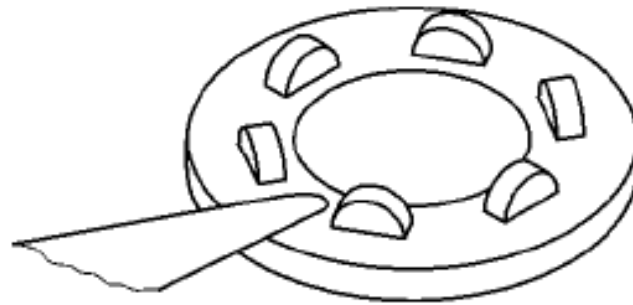
When all the assemblies in a connection have been snug tightened then final tightening can commence. Tightening shall be carried out progressively from the most rigid part of the connection to the least rigid part. Tightening shall continue until the specified indicator gap has been achieved (see Table 1).

Do not expect the indicator gaps to be equal around the circumference. Due to the tightening operation the direct tension indicator invariably pulls down more on one side than the other. The direct tension indicator is designed to accommodate this and the average gap will always give the correct tension.

<b>Table 1</b>	
<b>Thickness of feeler gauge</b>	
Direct tension indicator position	Thickness of feeler gauge
Under nut with a nut face washer, when nut is rotated (Figure 1)	0.25mm

The average specified indicator gap shall be determined using the following measurement procedure; the feeler gauge shall be used as a 'no go' inspection tool. The feeler gauge shall be pointed at the centre of the bolt see Figure 2 and shall refuse to enter the number of refusal spaces specified in Table 2. It is advisable to leave some small gap to prevent accidental overtightening and breakage but if the gap has been completely closed inadvertently this is not considered by Cooper & Turner to be cause for rejection.

<b>Table 2</b>	
<b>Feeler gauge requirements</b>	
Number of indicator protrusions	Minimum number of feeler gauge refusals
4	3
5	3
6	4
7	4
8	5
9	5



**Figure 2 - Checking the indicator gap (example with six protrusions)**

When the Direct Tension Indicators are installed in accordance with Cooper & Turner's instructions then the shank tension achieved will be in the range shown below in Table 3.

<b>Table 3</b>		
Nominal bolt diameter	Shank tension kN	
	H10 for 10.9	
	min	max
M16	110	132
M20	172	206
M22 <sup>1)</sup>	212	254
M24	247	296
M27 <sup>1)</sup>	321	385
M30	393	472
M36	572	688

<sup>1)</sup> Non – preferred sizes. Can only be supplied if the quantity required is sufficient to warrant manufacture.

**Inspection note:** The only way to verify that the shank tension has been achieved is by use of the appropriate feeler gauge as detailed above.

The calibrated torque wrench method is not sufficiently accurate for inspecting Direct Tension Indicator tightened assemblies, as it only measures resistance to turning. It should not be used in any circumstances.

**Dimensions of Holes for BS EN 14399-3 Assemblies**

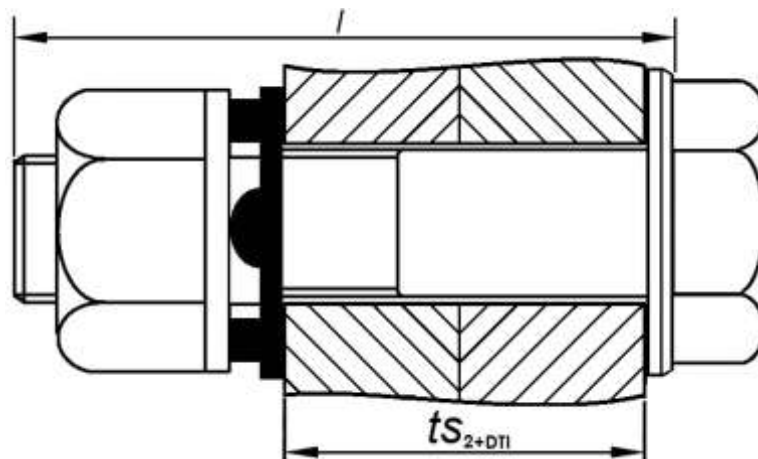
The nominal clearance for round holes is the difference between the nominal hole diameter and nominal bolt diameter. The hole sizes shown in Table 4 are in accordance with BS EN 1090-2:2008+ A1:2011

<b>Table 4</b>					
Nominal clearances for bolts (mm)					
Nominal bolt diameter (mm)	16	20	22	24	27 and over
Normal round holes (mm)	2				3
Tolerance on hole diameter $\pm 0.5\text{mm}$					

**Note:** Direct Tension Indicators will not function correctly if fitted against hole sizes larger than those specified above. If the connection design permits the use of oversize or slotted holes and these holes are present in an outer ply of the connection then a plate washer, having a normal clearance round hole, shall be fitted against the over size or slotted hole. Information regarding plate washers is contained in BS EN 1090-2:2008+ A1:2011.

**Grip lengths for BS EN 14399-3 PC 10.9 Assemblies with DTI's**

Table 5 below is provided for guidance only. Whilst this information is provided in good faith, no member of the Andaray group of companies shall be under any responsibility or liability in respect of errors or information that is found to be incorrect or for any reliance the user may place on it. The bolt thread lengths shown in BS EN 14399-3:2005 do not provide, for some of the shorter bolt lengths, the minimum four threads in the grip required by BS EN 1090-2. Cooper & Turner recognised this fact and have for a number of years manufactured certain of their BS EN 14399-3 assemblies with longer thread lengths to satisfy the minimum threads in the grip required by BS EN 1090-2. Table 5 therefore only applies to Cooper & Turner manufactured BS EN 14399-3 PC 10.9 assemblies with one BS EN 14399-6 washer fitted under the bolt head and a BS EN 14399-9 Nut Face Washer and a Direct Tension Indicator fitted under the nut.



**Figure 3 - DTI under nut with nut face washer and chamfered washer under bolt head.**

**Table 5**

Grip Lengths with one EN 14399-6 washer and one EN 14399-9 nut face washer and a DTI  $f_{S_2+DTI}$

Thread $d$	M16		M20		M22 <sup>1)</sup>		M24		M27 <sup>1)</sup>		M30		M36	
Bolt Length $l$	$f_{S_2+DTI \min}$ and $f_{S_2+DTI \max}$													
nominal	min	max	min	max	min	max	min	max	min	max	min	max	min	max
45	6	12												
50	9	17												
55	9	22	10	17	9	16								
60	20	27	14	22	9	21	12	18						
65	25	32	14	27	14	26	12	23	12	18				
70	30	37	24	32	14	31	17	28	12	23	16	21		
75	35	42	29	37	24	36	17	33	12	28	16	26		
80	40	47	34	42	29	41	17	38	20	33	16	31		
85	45	51	39	47	34	45	31	43	20	38	26	35	18	27
90	50	56	44	52	39	50	36	48	30	43	26	40	18	32
95	55	61	49	57	44	55	41	53	35	48	26	45	28	37
100	60	66	54	62	49	60	46	58	40	53	36	50	28	42
110	70	76	64	72	59	70	56	68	50	63	46	60	28	52
120	80	86	74	82	69	80	66	78	60	73	56	70	44	62
130	84	96	78	92	73	90	70	87	64	83	60	80	48	72
140	94	106	88	102	83	100	80	97	74	93	70	90	58	82
150	104	116	98	112	93	110	90	107	84	103	80	100	68	92
160	114	124	108	120	103	118	100	115	94	111	90	108	78	100
170							110	125	104	121	100	118	88	110
180							120	135	114	131	110	128	98	120
190							130	145	124	141	120	138	108	130
200							140	155	134	151	130	148	118	140

<sup>1)</sup> Non – preferred sizes. Can only be supplied if the quantity required is sufficient to warrant manufacture.

For the calculation of grip lengths  $f_{S_2+DTI}$  the following formulae have been used

$$f_{S_2+DTI \max} = l_{\min} - m_{\max} - 2h_{\max} - 1P - h_{2 \max}$$

$$f_{S_2+DTI \min} = l_{g \max} + 4P - 2h_{\min} - h_{1 \min}$$

where

$f_{S_2+DTI}$  grip length; the total thickness of the clamped parts between the nut bearing face and the bolt bearing face less the thickness of the two washer and the DTI (mm)

$l$  length of the bolt (mm)

$m$  height of the nut (mm)

$h$  washer thickness (chamfered and nut face washers) (mm)

$P$  thread pitch (mm)

$h_1$  DTI material thickness (mm)

$h_2$  DTI thickness over protrusions (mm)

$l_g$  distance from the bearing face to the first full form (full profile) thread (mm)

**NOTE: This table is provided for guidance only, see disclaimer at bottom of page**

### Corrosion

BS EN 14399-3 assemblies, bolts, nuts, washers and Direct Tension Indicators, will corrode if not properly protected. They must be kept in a clean dry and well ventilated store. It is important that only the number required for immediate installation are taken from the stores in order that none are allowed to lie about on site and deteriorate.

Painting should be carried out at an early stage after tightening. This is particularly important in marine or other corrosive environments.

The metallic coatings applied to structural fasteners are only intended to give temporary protection during storage and installation – early painting will still be required. Research has shown that susceptibility to stress corrosion cracking, environmentally induced brittle failure and hydrogen embrittlement increases with tensile strength.

The presence of a metallic coating lowers the threshold of the tensile strength at which these phenomena may occur. This means under certain conditions a metallic coating may reduce corrosive resistance, rather than enhance it and the specifier must take account of site environment in deciding the suitability or otherwise of a coating. Special care must be taken in the storage of these items and early protection by painting.

### Installation Tips

Check that all the bolts, nuts, washers and Direct Tension Indicators are the required property class / designation before fixing.

Ensure that the bolts fit the holes freely – forcing the bolts into the holes by means of hammering would damage the threads.

The Direct Tension Indicator protrusions must bear against the nut face washer (see Figure 1) – **NO OTHER WAY WILL DO.**

Hand wrenches may be satisfactory for tightening smaller diameters of bolts, but power tools or torque multipliers must be used for sizes above M20 diameter. It is most important that impact wrenches have adequate capacity to tighten bolts within about 15 seconds (slightly longer if gear driven wrenches used). Prolonged impacting can damage the bolt assembly to such an extent that the bolt may be fractured.

In selecting a tool, it is wise to choose one which has a torque output in excess of the theoretical figure required for tightening the largest bolt for which it will be needed, thus making some allowance for loss of performance due to wear, air leakage etc., and to help overcome the energy absorbed by higher than usual thread friction or 'springy' joints.

In order to obtain optimum tool performance the tool manufacturer's specification regarding air pressure, air flow volume and hose size must be followed.

The only way to confirm that a particular tool is suitable is by judging its performance in bolt tightening under actual site conditions.

Where steelwork is to be left exposed, tightening, inspection and painting should be carried out within the shortest time to prevent corrosion.

### **The Direct Tension Indicator is a precision made measuring device and on no account should it be given any further treatment after leaving the supplier**

A Direct Tension Indicator does not make an assembly any more difficult to tighten – it simply leaves a permanent witness that the assembly has been tightened to the correct tension.

### Problems tightening BS EN 14399-3 assemblies

The causes of the occasional problems encountered in trying to obtain the specified Direct Tension Indicator gaps when tightening BS EN 14399-3 assemblies can be generally categorised under three headings:-

**1) Fit** – Ensure that the bolts fit the holes freely and that poor alignment of the holes is not causing the bolts to trap.

**2) Tooling** – Using adequate capacity impact wrenches and satisfactory thread condition, the tightening operation should be completed within about 15 seconds (slightly longer if gear driven wrenches used). If it is not check for dry, rusty or damaged threads, poor fit, bad hole alignment or for a tool fault.

**3) Additional Lubrication** – The nuts which Cooper & Turner supply with their BS EN 14399-3 assemblies are lubricated but in certain circumstances e.g. if the assemblies have experienced less than ideal storage conditions, additional lubrication may be necessary.



## Use of High Strength Structural Bolting Assemblies for Preloading BS EN 14399-3 PC 10.9 Hex with Direct Tension Indicators (DTI's)

During the tightening of BS EN 14399-3 assemblies high frictional stress can develop and the likelihood of this occurring is greater when the assemblies have one of the thicker sacrificial coatings i.e. galvanizing or sherardizing, but it can occur occasionally with self colour products. In certain circumstances the frictional stress can lead to torsional failure of the bolts before the proof load is achieved. This high frictional stress can be reduced by the application of a suitable high pressure lubricant which should be applied both to the nut threads and the nut washer face. In Cooper & Turner's experience the most effective and economical lubricant for this purpose is tallow\*.

The use of a high pressure lubricant in the way described has no detrimental effect whatsoever on the functioning of the installed BS EN 14399-3 assemblies. The lubricant simply allows the torque applied to the nut to be more efficiently transferred into bolt shank tension and since the shank tension in the assembly is indicated by the Direct Tension Indicator gap, the reduction in the installation torque is of no consequence.

\* In the UK, tallow is generally available from plumbers' merchants. In parts of the world where tallow may not be available a product called 'stick wax', manufactured by a number of companies including Johnson and Castrol, is used for on-site lubrication of fasteners, alternatively a grease with a high molybdenum content may be effective. The high pressure lubricant selected should be of medium to high viscosity and should be applied both to the nut threads and nut washer face in order to prevent the possibility of contaminating the inner faces of the BS EN 14399-3 joint.

### Final Note

The recommendations detailed in this data sheet are a summary of our experience.

The final responsibility for the specification and the use of High Strength Structural Bolting Assemblies for Preloading BS EN 14399-3 Property Class 10.9 with Direct Tension Indicators must lie with the Designer, Consultant or Engineer, who must satisfy themselves that what they are specifying is what they themselves know from their own experience to be suitable, for the particular application for which they are responsible.